

Application of High-Order Discontinuous Galerkin Method to LES/DES Test Cases Using Computers with High Number of Cores

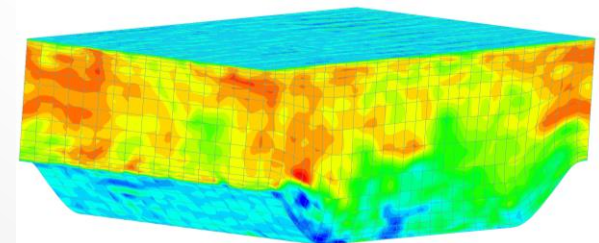
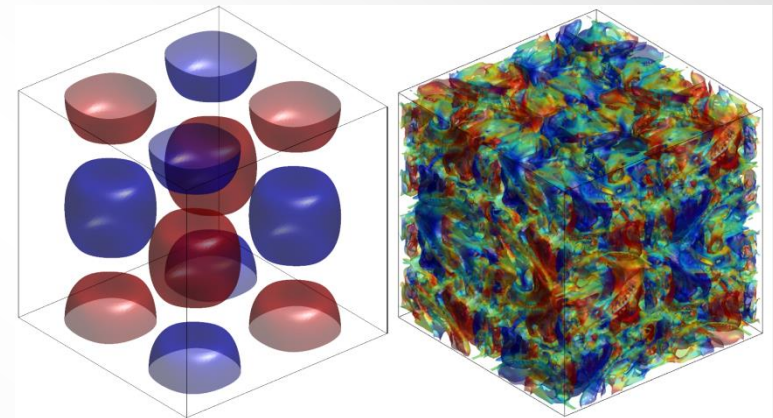
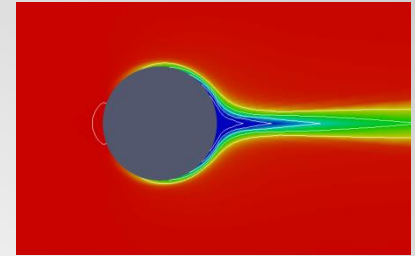
I. Bosnyakov^{1,2}, S. Mikhaylov^{1,2}, V. Podaruev^{1,2},
A. Troshin^{1,2}, V. Vlasenko^{1,2}, A. Wolkov^{1,2}

¹ Central Aerohydrodynamic Institute (TsAGI), Russia

² Moscow Institute of Physics and Technology (MIPT), Russia

Outline

- Introduction
- Discontinuous Galerkin and Finite Volume methods
- Preliminary tests
 - Flow over cylinder
 - Evolution of 2D vortex
- Base tests
 - Taylor–Green vortex
 - Periodic hill flow
- Nozzle test case (first results)
- Conclusions



Introduction

Towards Industrial LES/DNS in Aeronautics –
Paving the Way for Future Accurate CFD



- Objective: Development and testing of TsAGI code based on the high-order Discontinuous Galerkin Method (DG) for turbulent flow computations (aerodynamics, aeroacoustics)
- In addition, comparison with the Finite Volume Methods (FV – TsAGI's code) is presented

Finite Volume Method (implementation of TsAGI)

- Implementation of WENO is partial:
 - One-dimensional reconstruction
 - One quadrature point on the side of the cell
 - Central difference scheme for viscous terms

Class A, second order *)

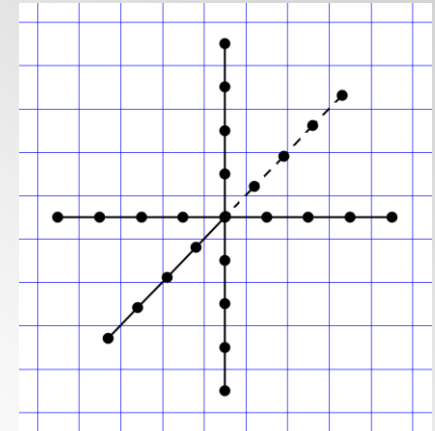
- In the following tests:
 - slope limiters are switched off (linear weights, no MP)
 - Roe Riemann solver is employed:

$$\mathbf{F}_{i+1/2} = \frac{1}{2}[\mathbf{F}(\mathbf{Q}_L) + \mathbf{F}(\mathbf{Q}_R)] - \frac{1}{2}\alpha(A^+ - A^-)(\mathbf{Q}_R - \mathbf{Q}_L);$$

$\alpha = 1 \rightarrow$ upwind scheme,

$\alpha = 0 \rightarrow$ central scheme

- Runge–Kutta, TVD3



*) Zhang R., Zhang M., Shu Ch. W. On the order of accuracy and numerical performance of two classes of finite volume WENO schemes // Communications in Computational Physics 9 (2011), No 3, pp. 807–827

Discontinuous Galerkin Method (1)

The system of equations and solution expansion:

$$\frac{\partial \mathbf{U}}{\partial t} + \nabla \cdot \mathbf{F}(\mathbf{U}, \mathbf{G}) = 0 \quad \mathbf{U}(\mathbf{x}, t) = \sum_{j=1}^{K_f} u_j(t) \varphi_j(\mathbf{x})$$

We multiply it by φ_i and integrate over the volume of cell Ω :

$$\int_{\Omega} \left(\frac{\partial \mathbf{U}}{\partial t} + \nabla \cdot \mathbf{F} \right) \varphi_i d\Omega = 0, \quad i = 1, \dots, K_f$$

Substituting the expansion of \mathbf{U} and taking into account the orthonormality of basis functions,

$$\int_{\Omega} \varphi_i \varphi_j d\Omega = \delta_{ij}$$

We arrive at the equation system for expansion coefficients u_i :

$$\frac{du_i}{dt} + \oint_{\Sigma} \hat{\mathbf{F}} \cdot \mathbf{n} \varphi_i d\Sigma = \int_{\Omega} \mathbf{F} \cdot \nabla \varphi_i d\Omega$$

Discontinuous Galerkin Method (2)

The resulting system of equations:

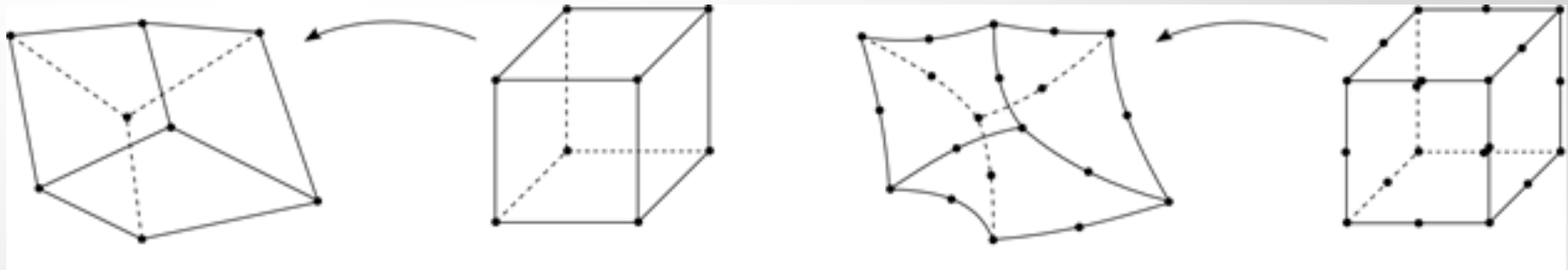
$$\frac{du_i}{dt} + \oint_{\Sigma} \hat{\mathbf{F}} \cdot \mathbf{n} \varphi_i d\Sigma = \int_{\Omega} \mathbf{F} \cdot \nabla \varphi_i d\Omega$$

Roe Riemann solver for inviscid flux

Bassi–Rebay 2 method for viscous flux

- $\{ \varphi_j(\mathbf{x}) \}$ is full orthonormal polynomial set up to order $K=1, 2, 3, 4, 5$
- integration is performed using the Gauss formula with tensor product of 1D Gauss–Legendre quadratures
- second order curvilinear meshes are used
- Runge–Kutta, SSP5

Elementary hexahedrons and their mappings



Serendipity transformations:
a) "linear", 8 vertices

b) "quadratic", 20 points

Barycenter coordinates:

$$x_0 = \frac{\int_{\Omega} x d\Omega}{\int_{\Omega} d\Omega}, \quad y_0 = \frac{\int_{\Omega} y d\Omega}{\int_{\Omega} d\Omega}, \quad z_0 = \frac{\int_{\Omega} z d\Omega}{\int_{\Omega} d\Omega}$$

$$\mathbf{I} = \begin{bmatrix} \int_{\Omega} (\tilde{y}^2 + \tilde{z}^2) d\Omega & -\int_{\Omega} \tilde{x}\tilde{y} d\Omega & -\int_{\Omega} \tilde{x}\tilde{z} d\Omega \\ -\int_{\Omega} \tilde{x}\tilde{y} d\Omega & \int_{\Omega} (\tilde{x}^2 + \tilde{z}^2) d\Omega & -\int_{\Omega} \tilde{y}\tilde{z} d\Omega \\ -\int_{\Omega} \tilde{x}\tilde{z} d\Omega & -\int_{\Omega} \tilde{y}\tilde{z} d\Omega & \int_{\Omega} (\tilde{x}^2 + \tilde{y}^2) d\Omega \end{bmatrix}$$

symmetry and
positive definiteness
properties

$$\tilde{x} = x - x_0$$

$$\tilde{y} = y - y_0$$

$$\tilde{z} = z - z_0$$

Unit and mutually orthogonal eigenvectors of inertia tensor \mathbf{I} is: $\mathbf{e}_1 \mathbf{e}_2 \mathbf{e}_3$

$$\begin{bmatrix} x_{\Omega} \\ y_{\Omega} \\ z_{\Omega} \end{bmatrix} = \begin{bmatrix} e_{11} & e_{12} & e_{13} \\ e_{21} & e_{22} & e_{23} \\ e_{31} & e_{32} & e_{33} \end{bmatrix} \begin{bmatrix} \tilde{x} \\ \tilde{y} \\ \tilde{z} \end{bmatrix}$$

Basis functions: $\psi_j(\mathbf{x}_{\Omega}) = s_j^{-1} x_{\Omega}^{\alpha_j} y_{\Omega}^{\beta_j} z_{\Omega}^{\gamma_j}$, $\alpha_j, \beta_j, \gamma_j \in \mathbf{Z}_+$, $0 \leq \alpha_j + \beta_j + \gamma_j \leq K$.

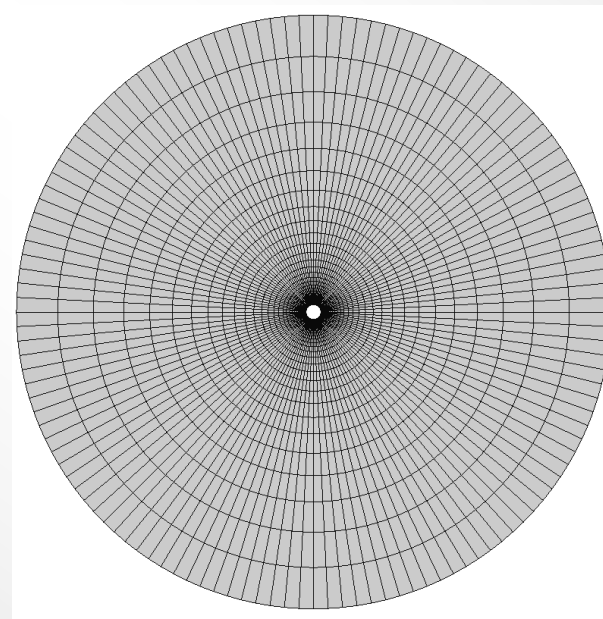
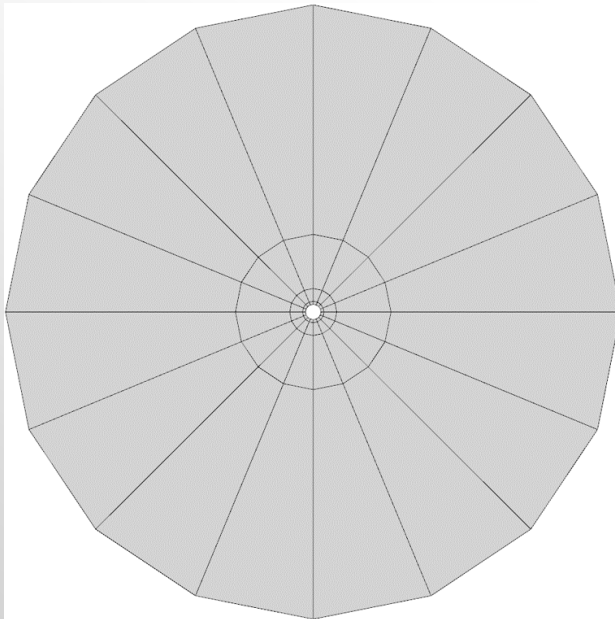
$$s_j = \sqrt{\int_{\Omega} (x_{\Omega}^{\alpha_j} y_{\Omega}^{\beta_j} z_{\Omega}^{\gamma_j})^2 d\Omega}$$

$$\int_{\Omega} \psi_j^2(\mathbf{x}_{\Omega}) d\Omega = 1$$

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Flow over cylinder: computational mesh and flow parameters

- A series of refined meshes with dimensions from $16 \times 4 \times 1$ to $128 \times 32 \times 1$ cells
- $R_{\text{cylinder}} = 0.5$, $R_{\text{outer}} = 20$, $\Delta z = 0.1$
- Cylinder surface is «slip wall», side planes are «symmetry»
- Freestream values are imposed at the outer boundary
- Freestream Mach number $M_{\infty} \approx 0.15$



Flow over cylinder: total pressure field

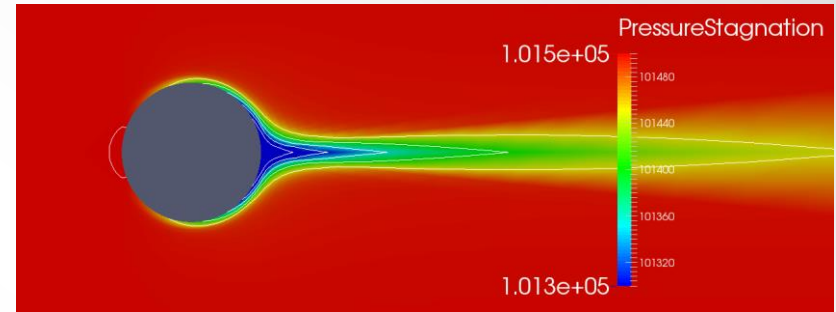
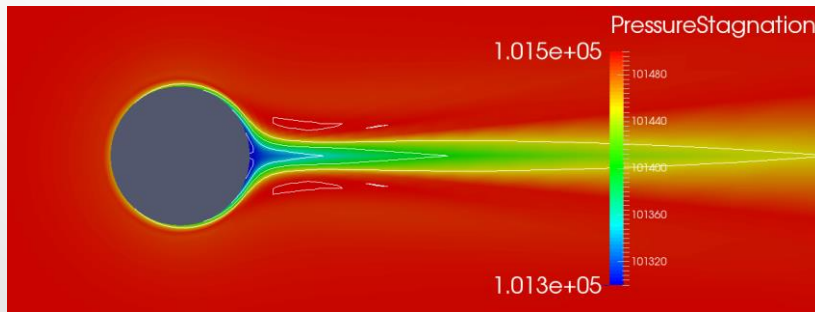
- 128 x 32 x 1 mesh

polynomial order



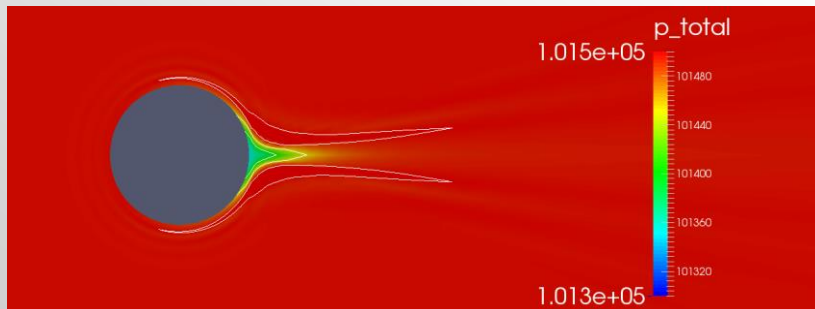
FV, central scheme

DG K = 1



FV, WENO 5

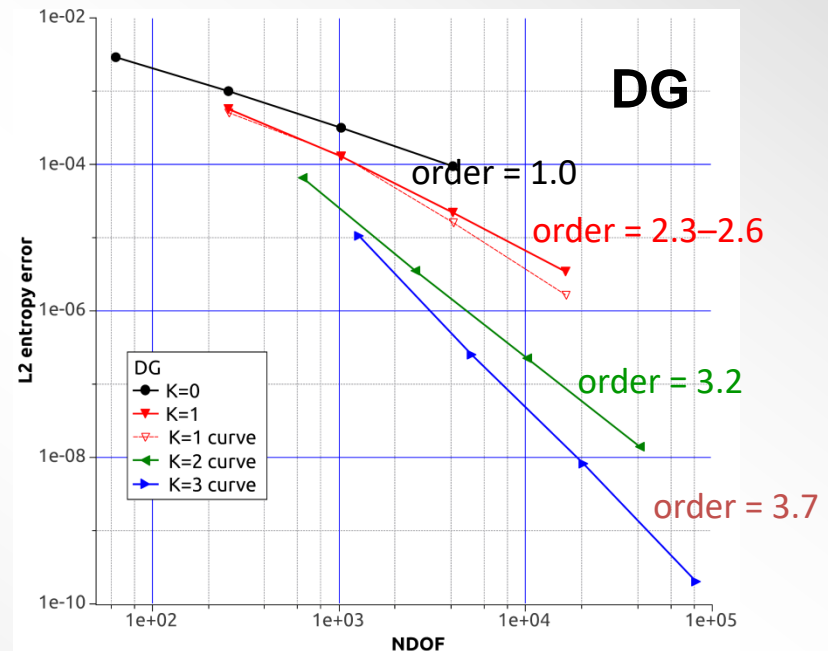
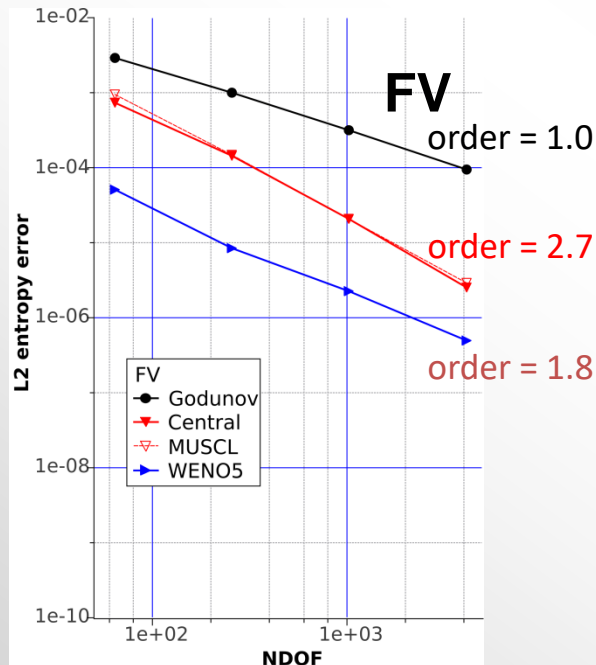
DG K = 3 on a curved mesh



Flow over cylinder: entropy error, L2 norm

$$e_{entropy} = \left(\frac{p}{p_\infty} \right) / \left(\frac{\rho}{\rho_\infty} \right)^\kappa - 1, \quad Order = 2 \frac{\log(e_{i-1} / e_i)}{\log(NDOF_i / NDOF_{i-1})}.$$

NDOF = (Number of cells) x (Number of basis functions)



- With FV, error 10^{-10} can be achieved on a mesh of size 10^{10} DOFs
- DG requires only 10^5 DOFs for the same accuracy

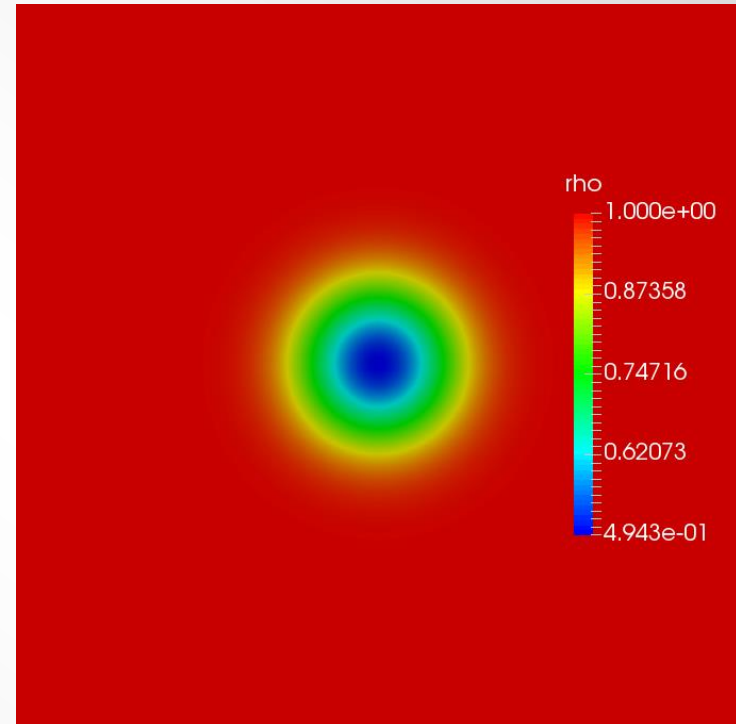
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Evolution of 2D vortex:

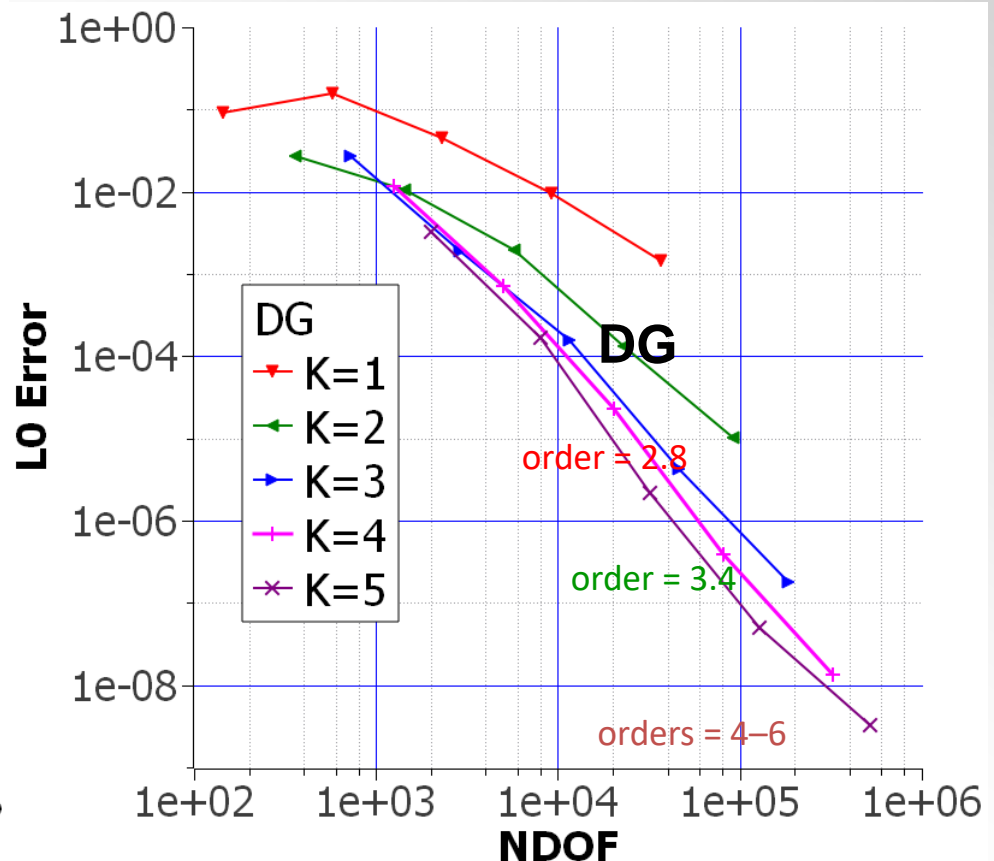
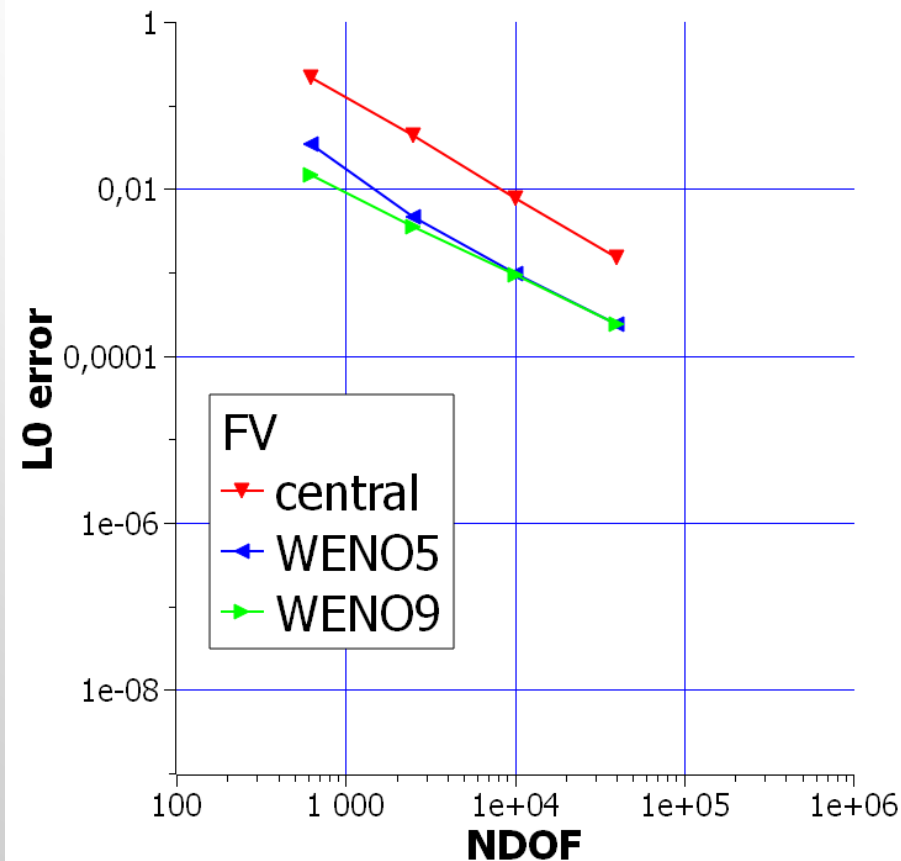
$$u = 1 - \frac{\varepsilon}{2\pi} e^{\frac{1}{2}(1-r^2)} y, \quad v = 1 - \frac{\varepsilon}{2\pi} e^{\frac{1}{2}(1-r^2)} x,$$

$$T = 1 - \frac{(\gamma - 1)\varepsilon^2}{8\gamma\pi^2} e^{(1-r^2)}, \quad \frac{p}{\rho^\gamma} = 1,$$

where $r^2 = x^2 + y^2$, $\varepsilon = 5$



Evolution of 2D vortex: L0 error

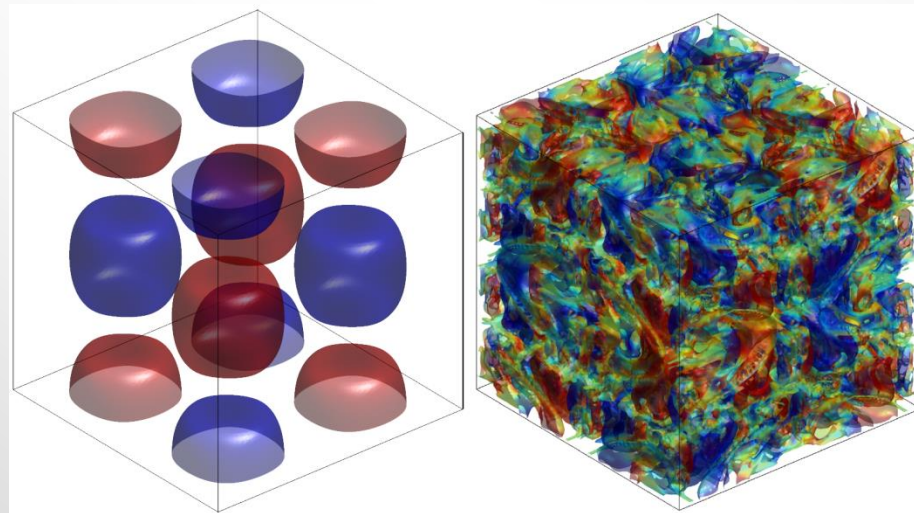


- With FV, error 10^{-8} can be achieved on a mesh of size 10^{10} DOFs
- DG requires only 10^5 DOFs for the same accuracy

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Taylor–Green Vortex test case

$$\begin{aligned}
 u &= V_0 \sin\left(\frac{x}{L}\right) \cos\left(\frac{y}{L}\right) \cos\left(\frac{z}{L}\right), \\
 v &= -V_0 \cos\left(\frac{x}{L}\right) \sin\left(\frac{y}{L}\right) \cos\left(\frac{z}{L}\right), \\
 w &= 0, \\
 p &= p_0 + \frac{\rho_0 V_0^2}{16} \left(\cos\left(\frac{2x}{L}\right) + \cos\left(\frac{2y}{L}\right) \right) \left(\cos\left(\frac{2z}{L}\right) + 2 \right)
 \end{aligned}$$



Pressure isosurfaces, Re = 1600

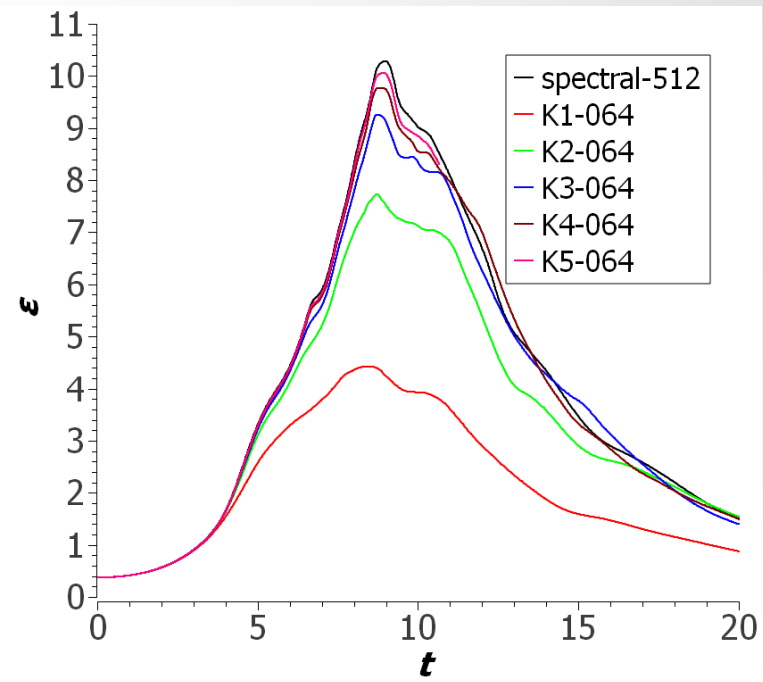
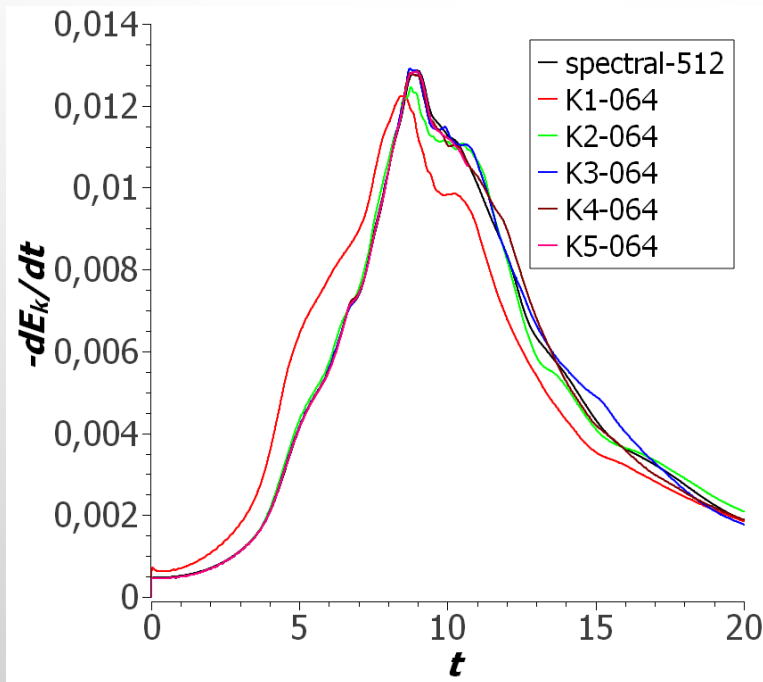
Taylor–Green Vortex: DG method accuracy, 64^3 mesh

Turbulent kinetic energy

$$E_k = \frac{1}{\rho_0 \Omega} \int_{\Omega} \rho \frac{v \cdot v}{2} d\Omega$$

Enstrophy

$$\epsilon = \frac{1}{\rho_0 \Omega} \int_{\Omega} \rho \frac{\omega \cdot \omega}{2} d\Omega$$



Spectral method reference data: W.M. van Rees, A. Leonard, D.I. Pullin, P. Koumoutsakos.

A comparison of vortex and pseudo-spectral methods for the simulation of periodic vortical flows at high Reynolds number // J. Comput. Phys. 230 (2011), pp. 2794–2805

Taylor–Green Vortex: convergence and time requirements

- NDOF = number of degrees of freedom
- t_{comp} = computing time for each calculation (scaled to 512 core cluster)
- **error** = enstrophy maximum difference obtained in the calculation and in the reference solution

FV schemes

| | 64 ³ | 96 ³ | 128 ³ | 192 ³ | 256 ³ | 384 ³ | 512 ³ |
|---------|--|--|--|---------------------------------------|---------------------------------------|---------------------------------------|--|
| central | | | 2.1 x 10 ⁶ 0.36 h 68% | | | | |
| WENO5 | | | 2.1 x 10 ⁶ 0.49 h 45% | | | | |
| WENO9 | 2.6 x 10 ⁵ 0.03 h 63% | 8.8 x 10 ⁵ 0.13 h 50% | 2.1 x 10 ⁶ 0.56 h 38% | 7.1 x 10 ⁶ 2.3 h 23% | 1.7 x 10 ⁷ 9.6 h 16% | 5.7 x 10 ⁷ 39 h 8.0% | 1.3 x 10 ⁸ 153 h 4.7% |

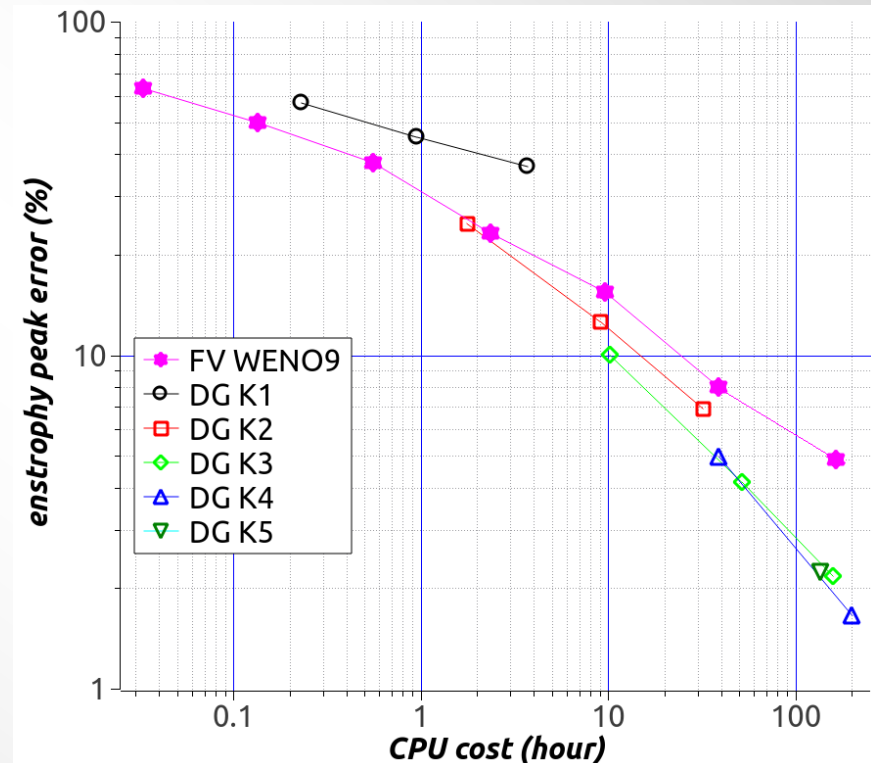
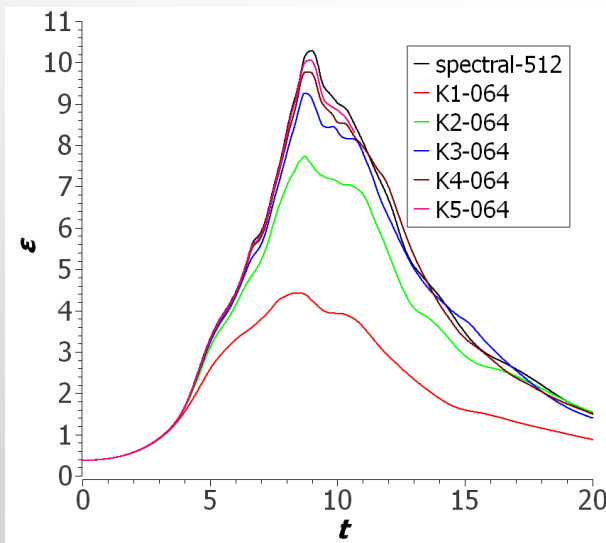
DG schemes

| | 64 ³ | 96 ³ | 128 ³ |
|-------|--|--|---|
| K = 1 | 1.0 x 10 ⁶ 0.23 h 60% | 3.5 x 10 ⁶ 1.0 h 45% | 8.4 x 10 ⁶ 3.7 h 37% |
| K = 2 | 2.6 x 10 ⁶ 1.8 h 25% | 8.9 x 10 ⁶ 9.1 h 13% | 2.1 x 10 ⁷ 32 h 6.9% |
| K = 3 | 5.2 x 10 ⁶ 10 h 10% | 1.8 x 10 ⁷ 52 h 4.2% | 4.2 x 10 ⁷ 159 h 2.2% |
| K = 4 | 9.2 x 10 ⁶ 39 h 5.0% | 3.1 x 10 ⁷ 198 h 1.7% | 7.3 x 10 ⁷ 623 h 0.89% |
| K = 5 | 1.5 x 10 ⁷ 136 h 2.2% | | |

- *Up to 4096 cores were used for DG computations*

Taylor–Green Vortex: enstrophy peak evaluation

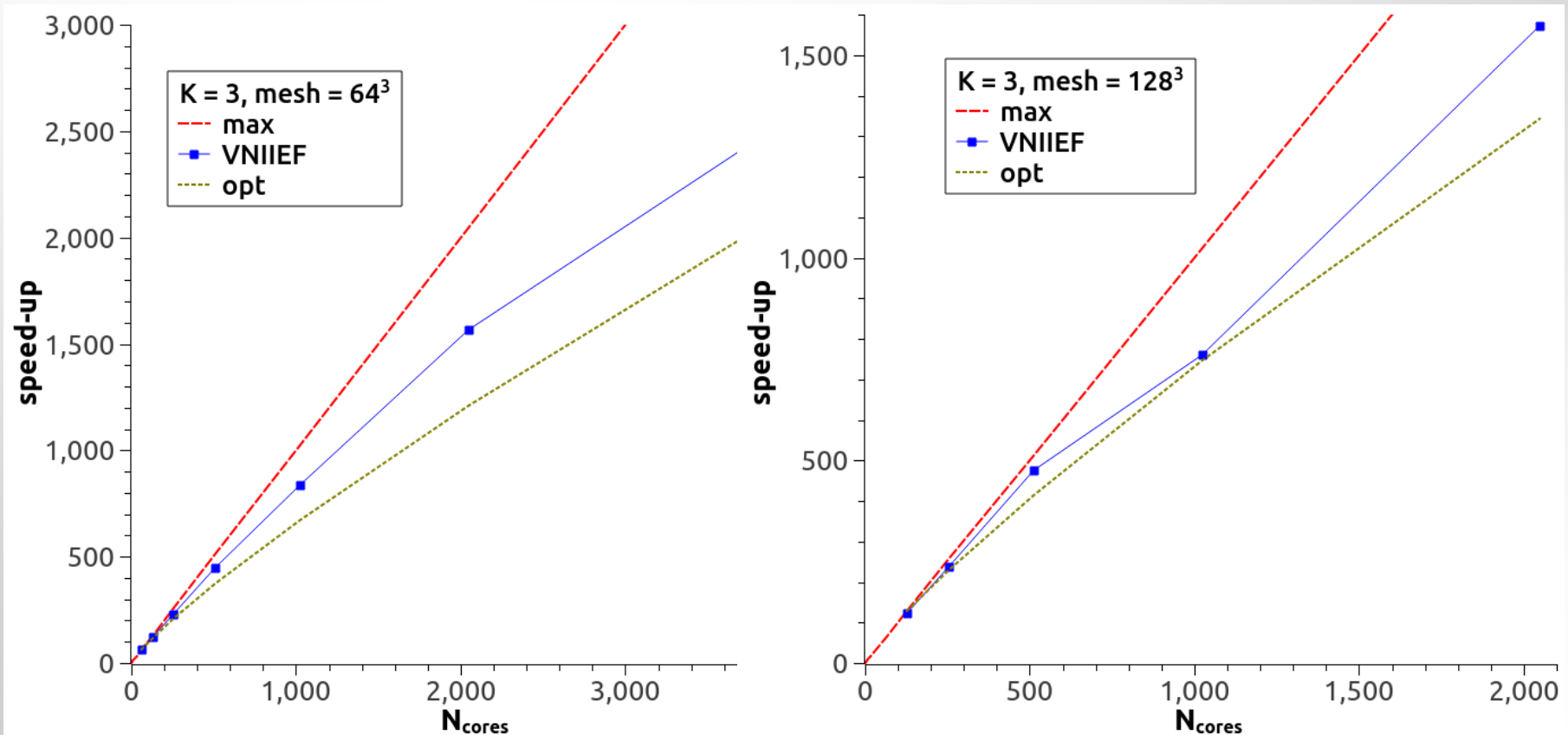
$$\epsilon = \frac{1}{\rho_0 \Omega} \int_{\Omega} \rho \frac{\omega \cdot \omega}{2} d\Omega$$



After $K > 2$ increase in the order of the scheme and increase in the computational grid size have virtually equal effect on enstrophy error level

Taylor–Green Vortex: MPI scalability

- max – maximum possible acceleration ($\text{max} = N_{\text{cores}}$)
- opt – acceleration 1.8 times for every doubling of CPU cores



Increase in the number of cores (> 4,000) leads to reduction in scalability

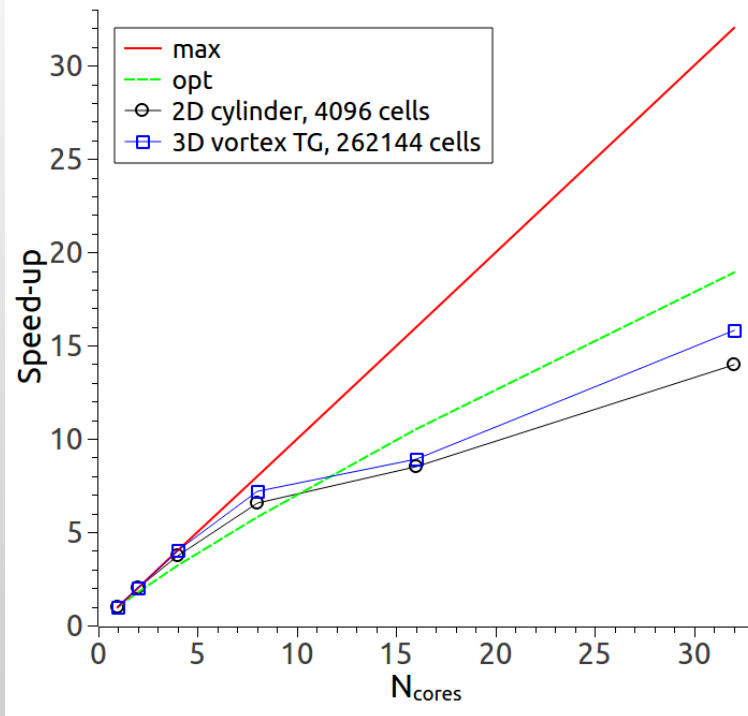
Taylor–Green Vortex: OpenMP scalability

MPI - Separated memory for each core and a big data exchanges;

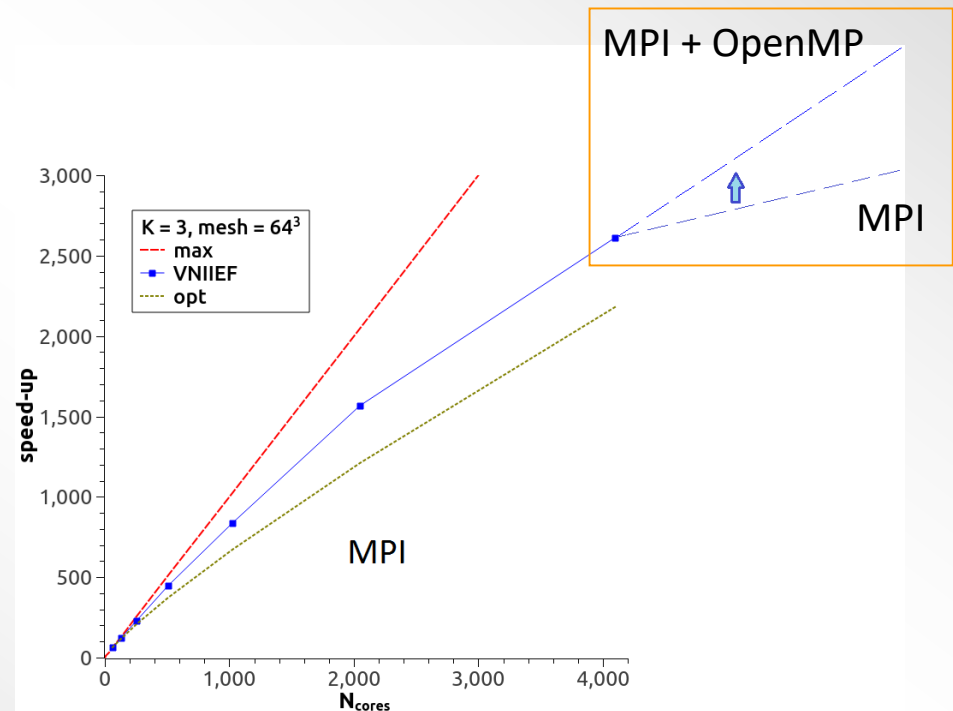
OpenMP - Shared memory for all cores of the computer node;

TsAGI cluster: 32 CPU cores on each computer node -> 8 CPU cores can be joint into one 8-thread process without loss of computational efficiency

K=3



forecast:

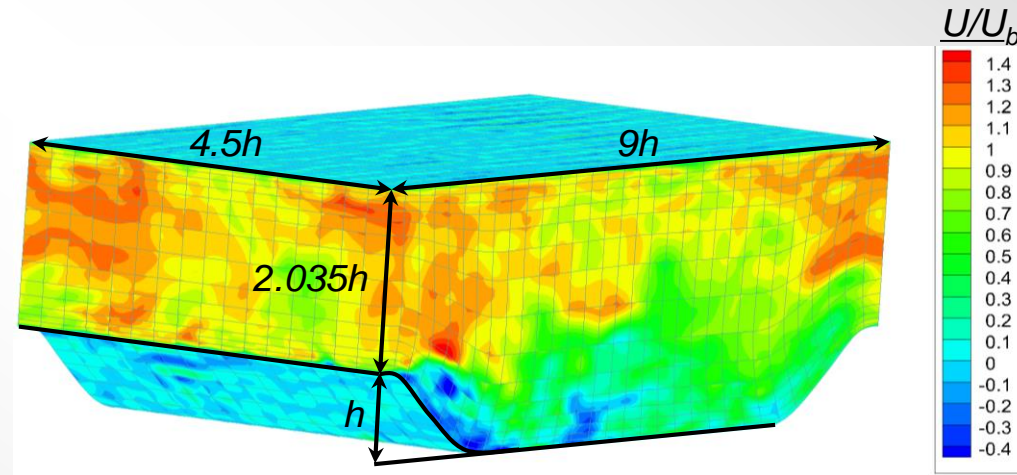


- MPI + OpenMP approach is promising with further increase of core number

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Periodic hill flow

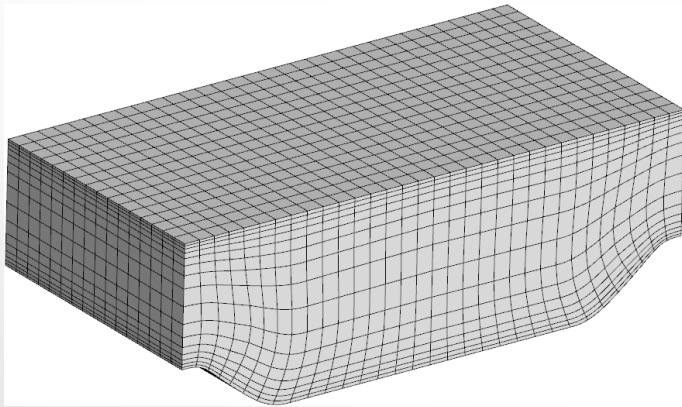
- An ERCOFTAC QNET CFD UFR 3-30 test case



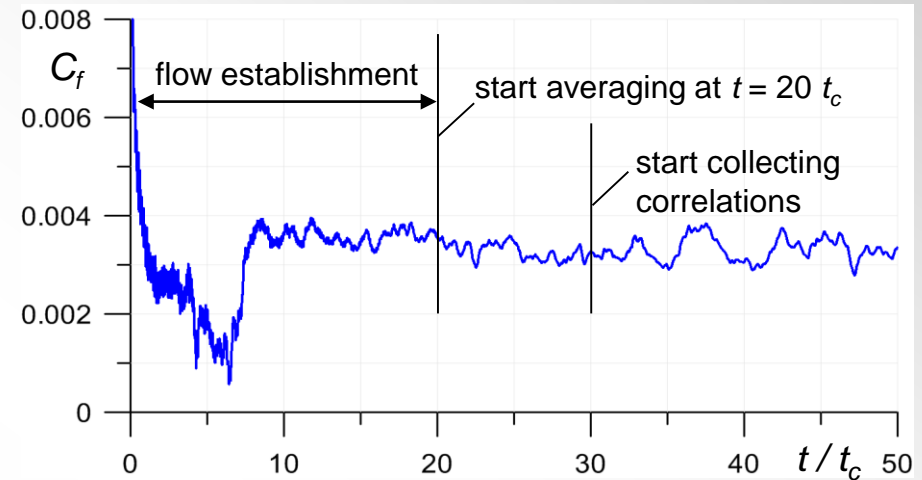
- streamwise and spanwise periodic flow
- forcing term dp/dx is imposed to maintain the mass flow rate
- Reynolds number $Re = 10595$, Mach number $M \approx 0.1$
- uniform initial flowfield, initial state is “forgotten”
- Implicit Large Eddy Simulation (ILES) based on DG $K = 1, 2, 3$

Periodic hill flow: computational mesh and averaging

relatively coarse 32 x 16 x 16 mesh
has been used



Averaging method



- The following data are collected:
 - Average velocity, pressure, density fields $U, V, W, P, \bar{\rho}$
 - Correlations (at the moment, in cell centers only $\overline{u'^2} = \overline{(u-U)^2}$, $\overline{v'^2}$, $\overline{w'^2}$, $\overline{u'v'}$, $\overline{u'w'}$, $\overline{v'w'}$
- Averaging is done over time (for at least 15 t_c) and over span (z axis direction)

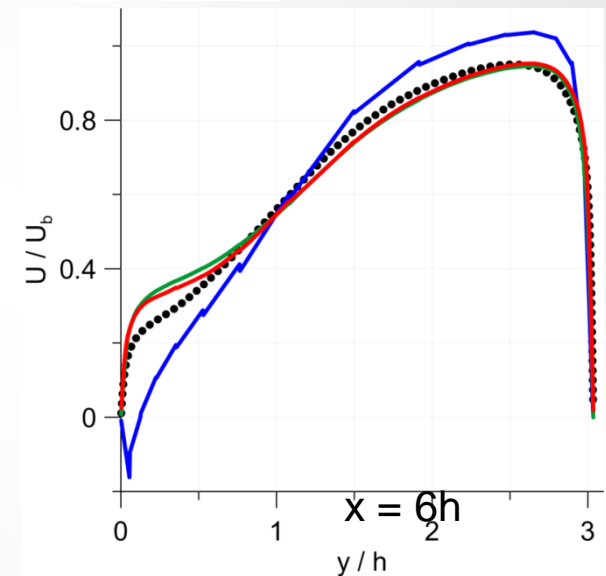
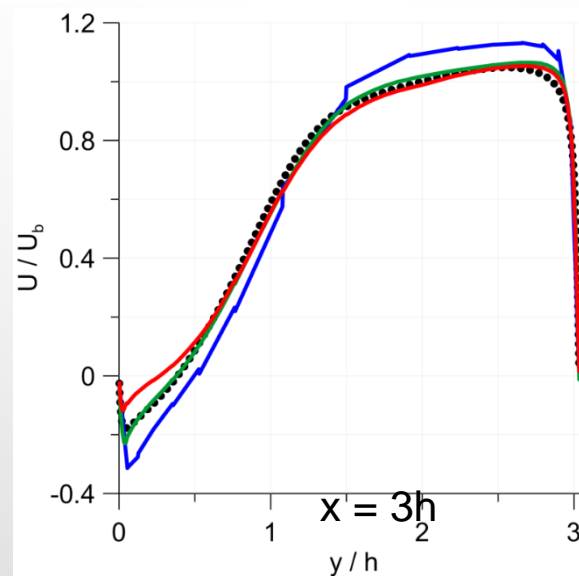
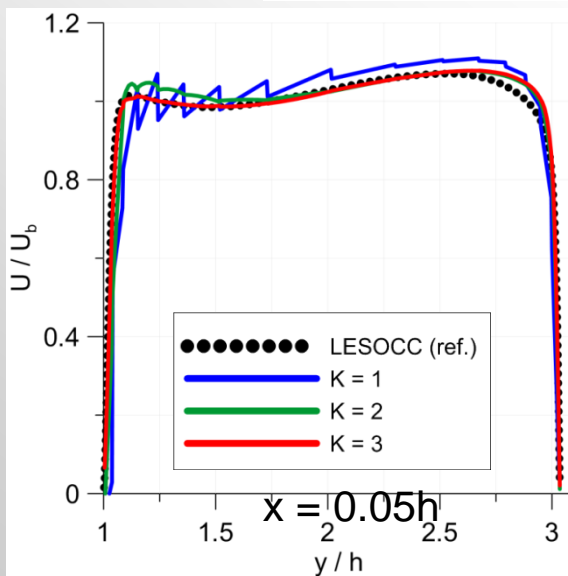
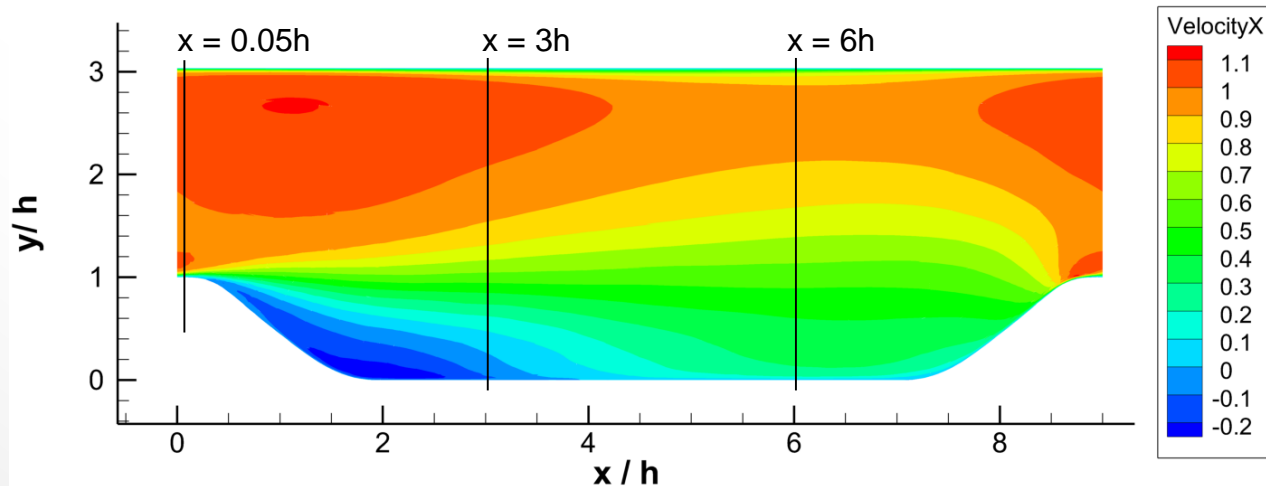
Reference solution: NDOF = 13,100,000;

DG solution: K=1 – NDOF=32*16*16*4=32,768

K=2 81,920

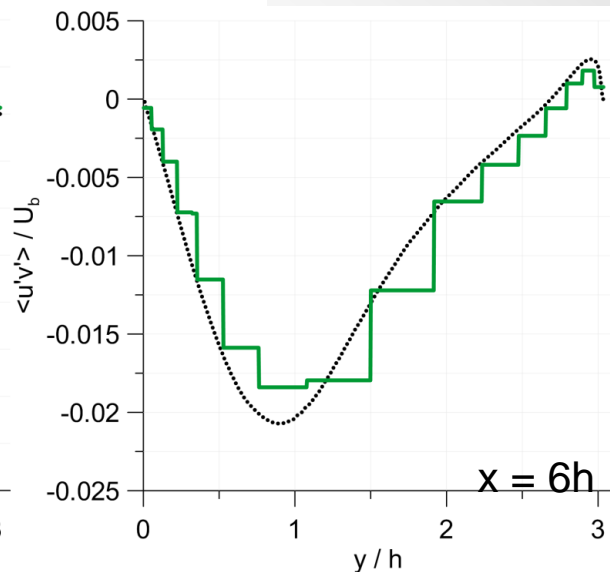
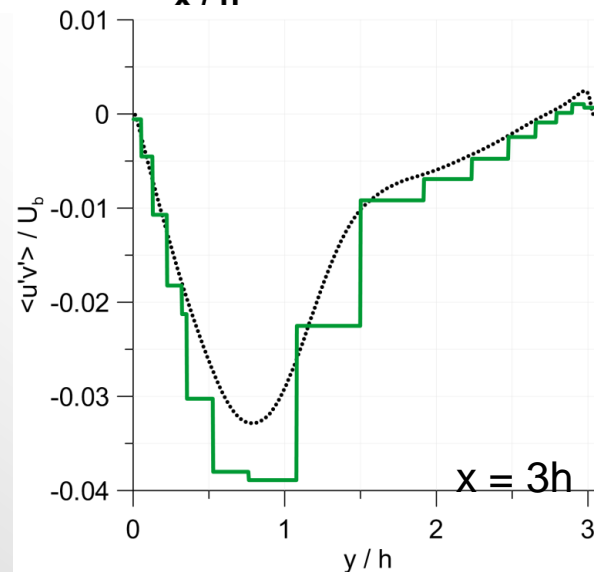
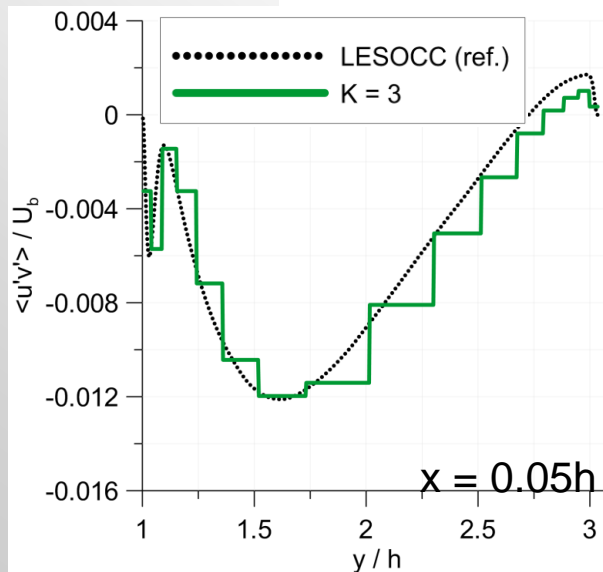
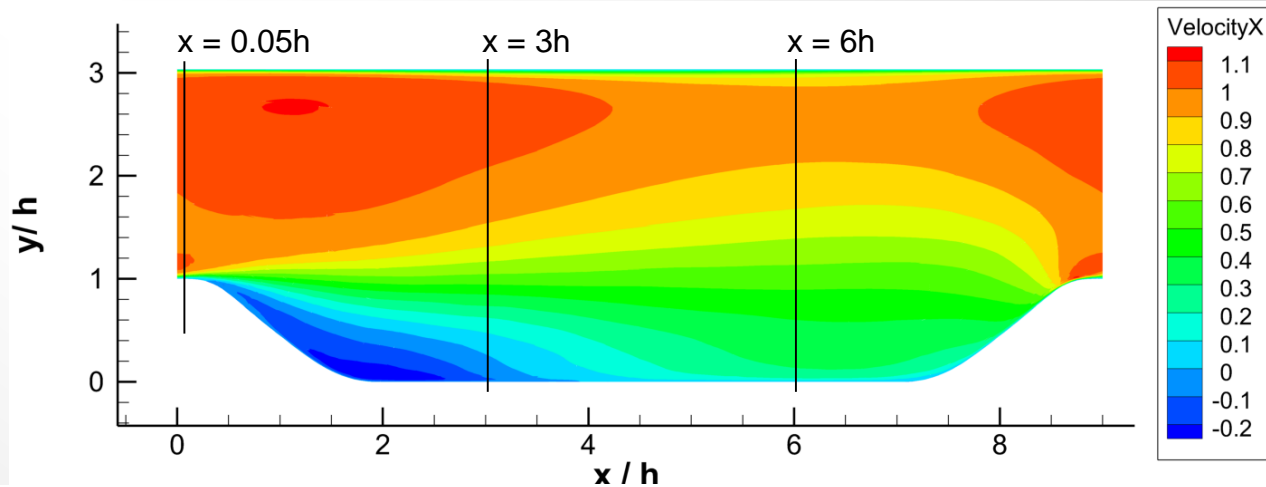
K=3 163,840

Periodic hill flow: mean velocity profiles



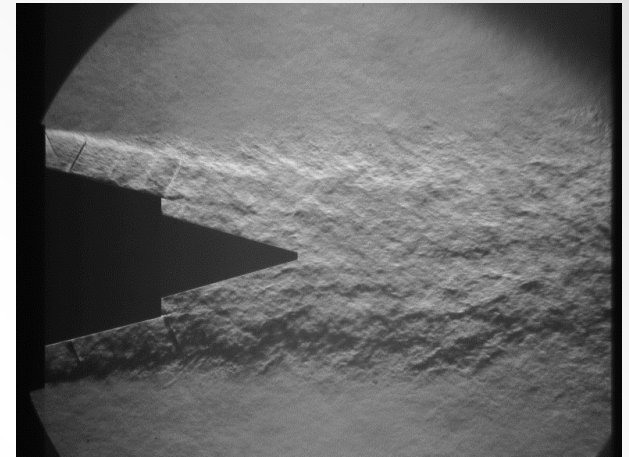
- Reference LES data: M. Breuer, N. Peller, Ch. Rapp, M. Manhart, Comput. Fluids 2009

Periodic hill flow: shear stress profiles



- Reference LES data: M. Breuer, N. Peller, Ch. Rapp, M. Manhart, Comput. Fluids 2009

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Detached-eddy Simulation DDES

P.R. Spalart, S. Deck, M.L. Shur, K.D. Squires, M.Kh. Strelets, A. Travin. A new version of detached-eddy simulation, resistant to ambiguous grid densities // Theor. Comput. Fluid Dyn. **20**, pp. 181–195, 2006

- A modified SA equation of the turbulence model is solved :

$$\frac{\partial \tilde{\nu}}{\partial t} + u_j \frac{\partial \tilde{\nu}}{\partial x_j} - \frac{\partial}{\partial x_j} \left(\frac{\nu + \tilde{\nu}}{\text{Pr}_t^{\tilde{\nu}}} \frac{\partial \tilde{\nu}}{\partial x_j} \right) = P_{\tilde{\nu}}(\dots, \boxed{\tilde{d}}) - D_{\tilde{\nu}}(\dots, \boxed{\tilde{d}})$$

- The length scale varies smoothly from d_{wall} (RANS) to Δ_{cell} (LES):

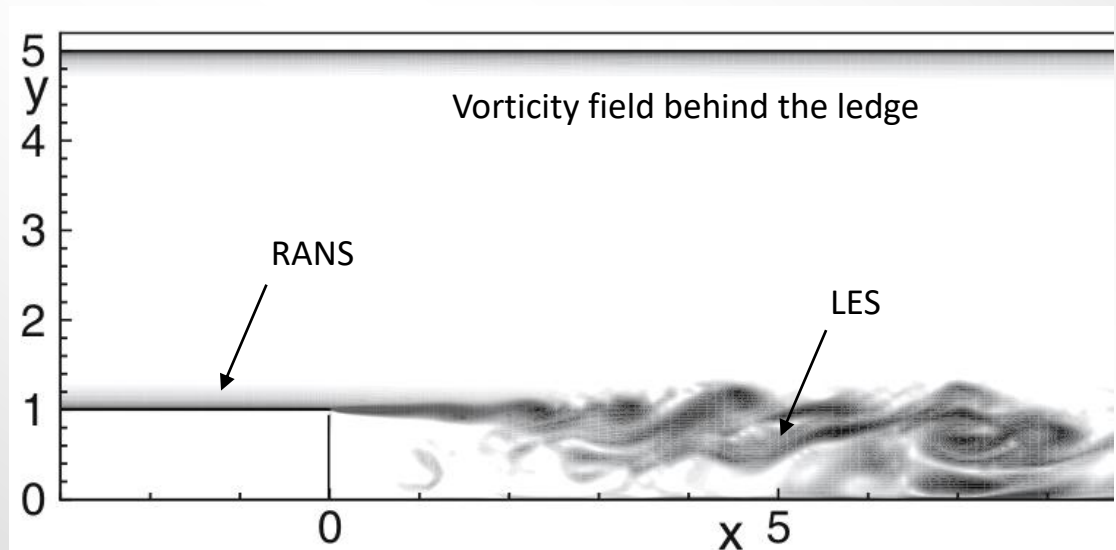
$$\tilde{d} = d_{\text{wall}} - f_d \max(0, d_{\text{wall}} - C_{\text{DES}} \Delta_{\text{cell}})$$

$$f_d = 1 - \text{th}(8r_d)^3$$

$$r_d = \frac{\nu + \nu_t}{\sqrt{\frac{\partial u_i}{\partial x_j} \frac{\partial u_i}{\partial x_j}} K^2 d_{\text{wall}}^2}$$

$$\Delta = \max(h_x, h_y, h_z)$$

$$C_{\text{DES}} = 0.65, K = 0.41$$



“Noise suppressing nozzle” test case TC-P4: Dual-stream jet nozzle

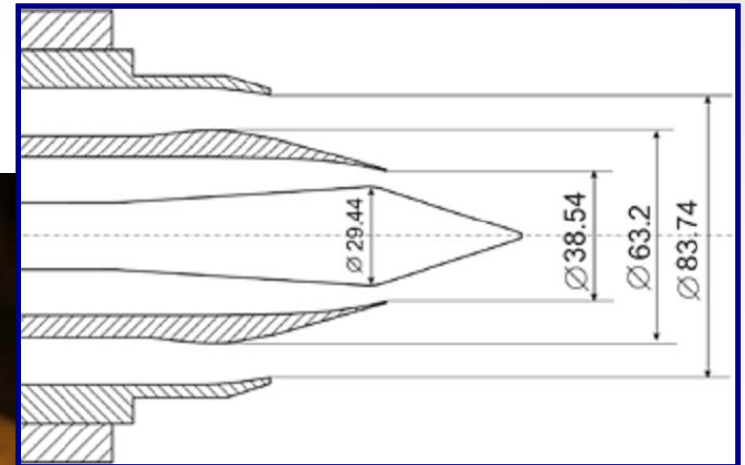


front view

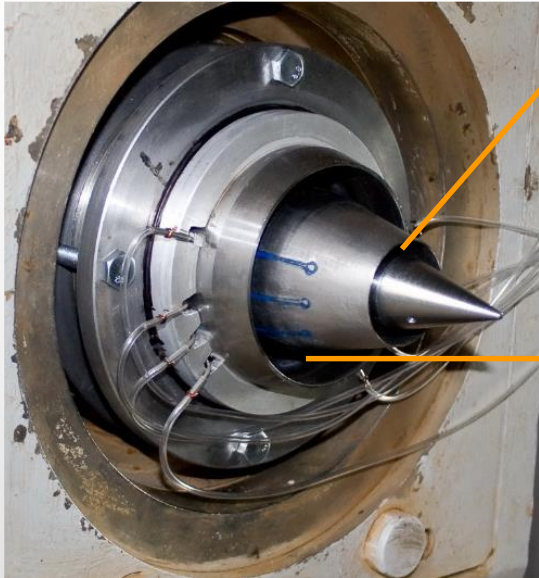
- dual-stream coaxial nozzle
- central body
- cold air flow
- pressure difference between the contours is generated by the grids



back view



Flow regime and visualization

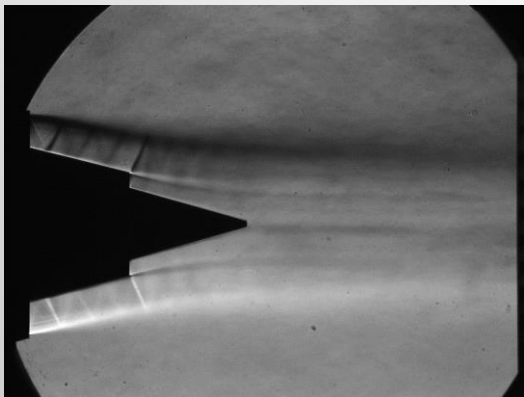


Inner contour:

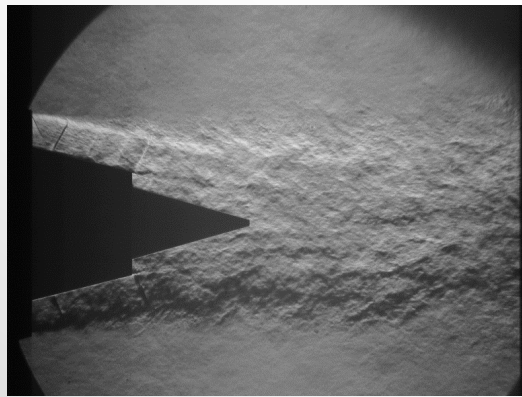
- subsonic jet, $M = 0.85$ at nozzle exit
- nozzle pressure ratio $\text{NPR}_1 = 1.72$
- diameter-based Reynolds number $\text{Re}_{1D} = 0.96 \cdot 10^6$

Outer contour:

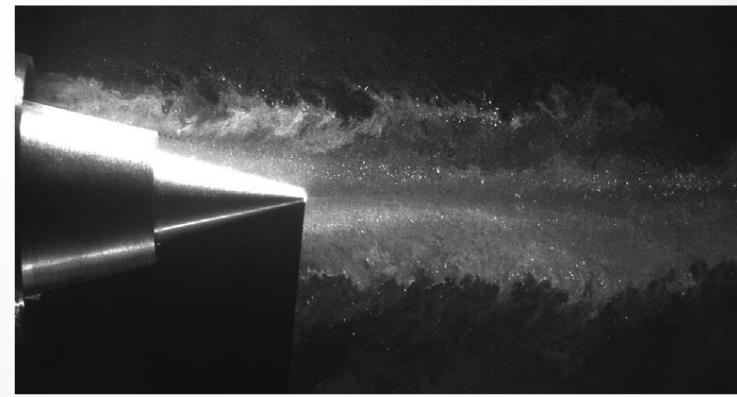
- supersonic underexpanded jet, $M=1$ at nozzle exit
- nozzle pressure ratio $\text{NPR}_2 = 2.25$
- diameter-based Reynolds number $\text{Re}_{2D} = 2.872 \cdot 10^6$



Shlieren visualization,
0.01 s exposure

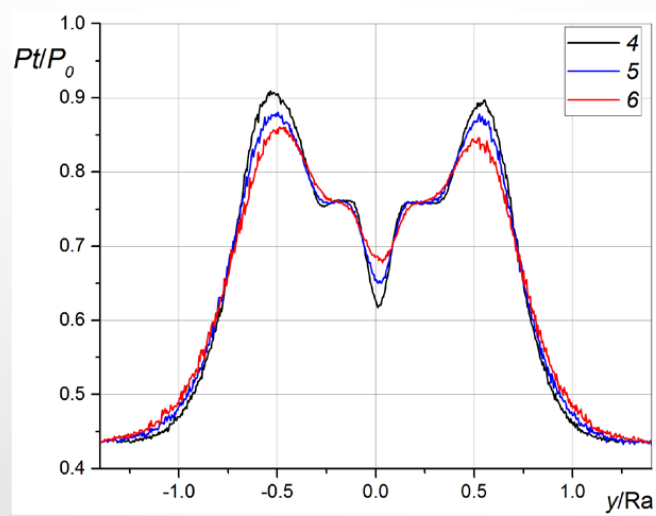
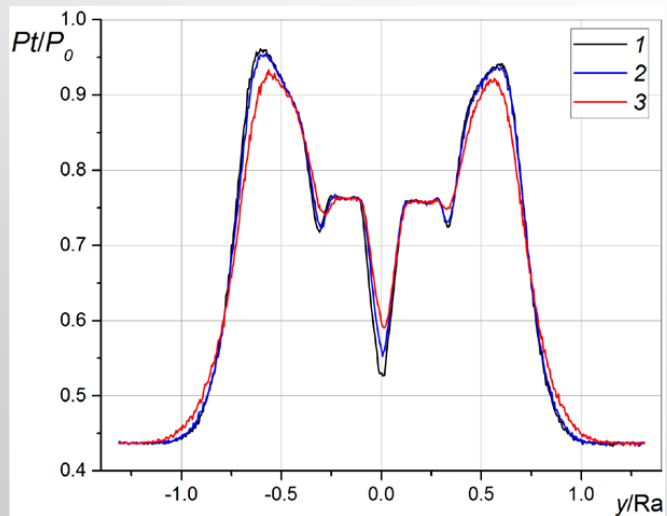
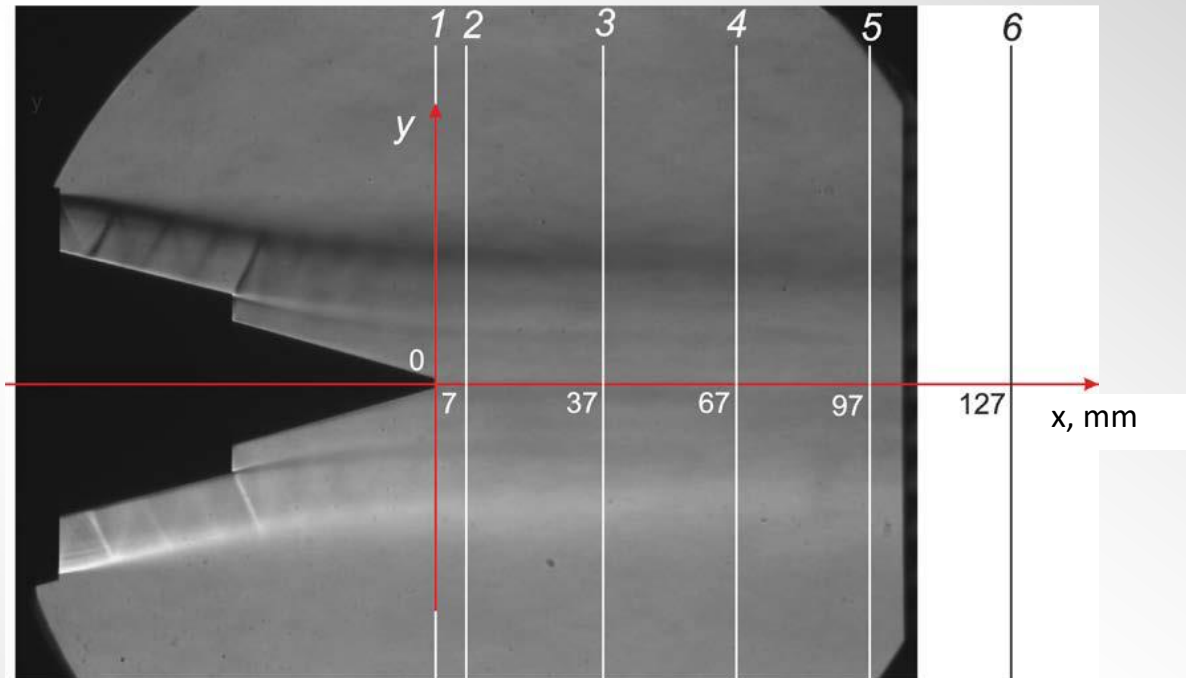


Shlieren visualization,
 $3 \cdot 10^{-6}$ s exposure

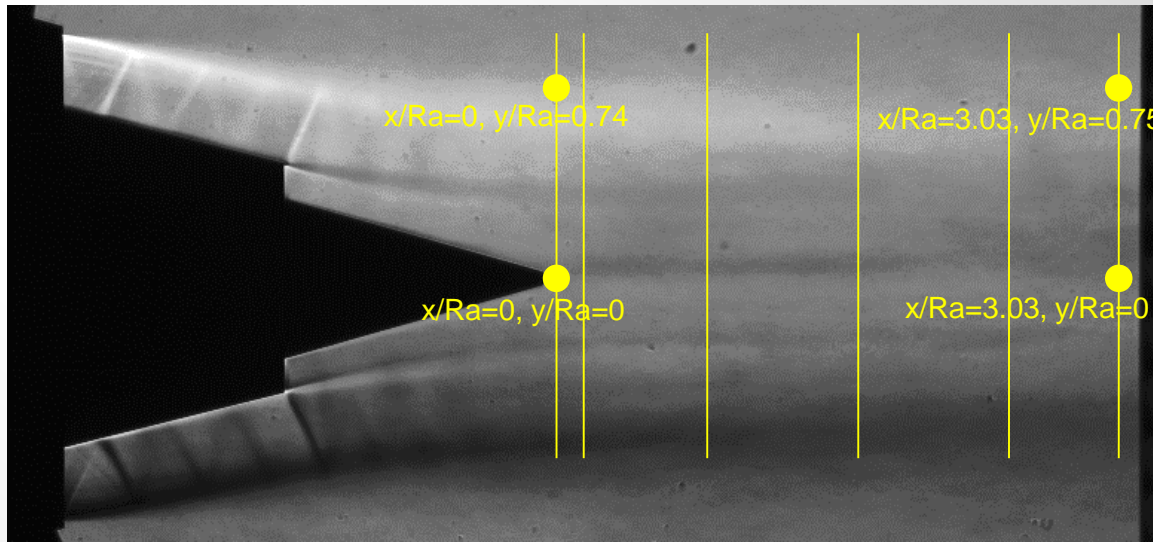


Laser knife visualization

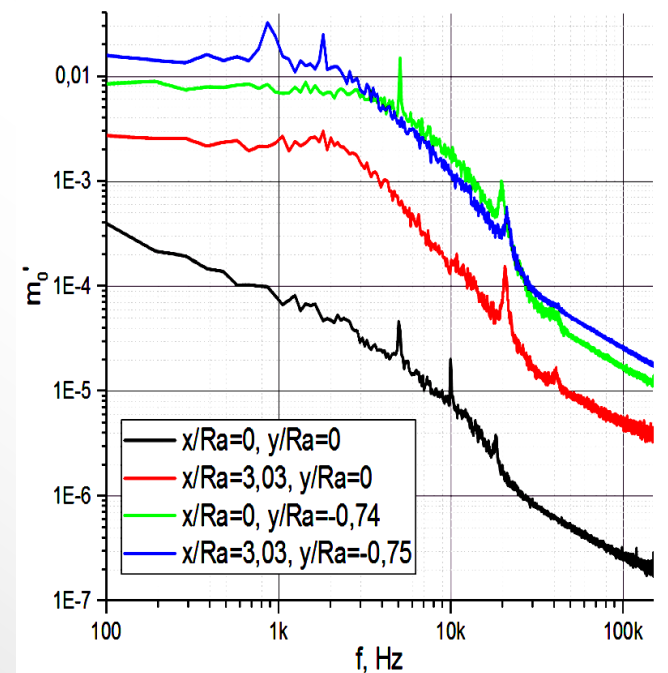
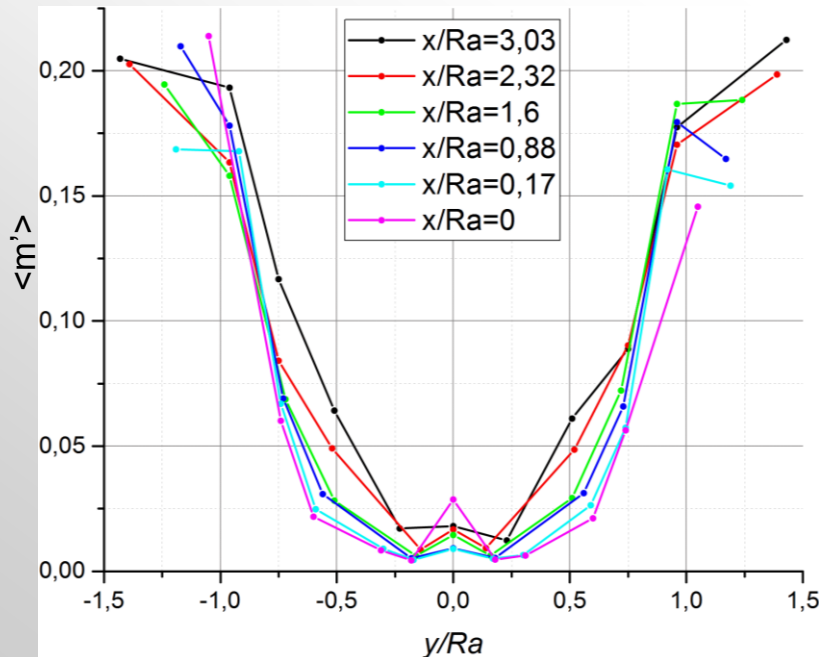
Pitot pressure measurements



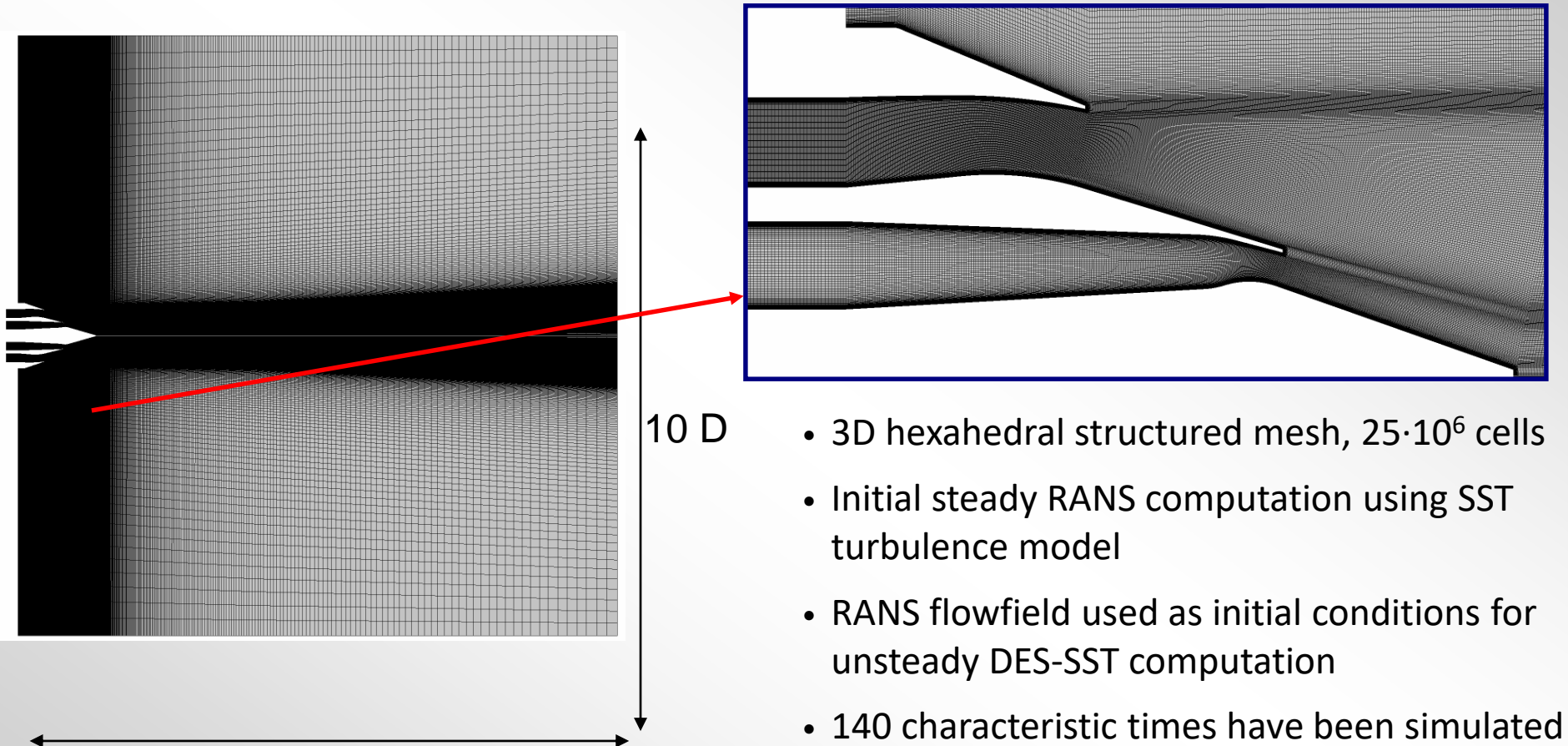
Root mean square mass flow rate pulsations at different jet cross sections and frequency amplitude spectrum was observed using hot wire



$$\langle m' \rangle = \frac{(\rho U)'}{\rho U}$$



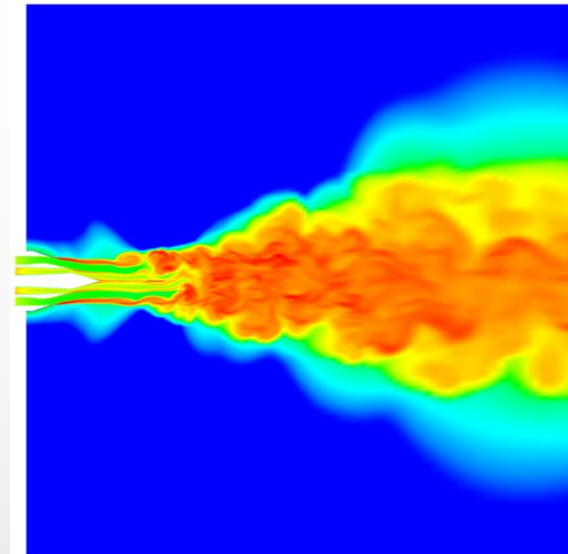
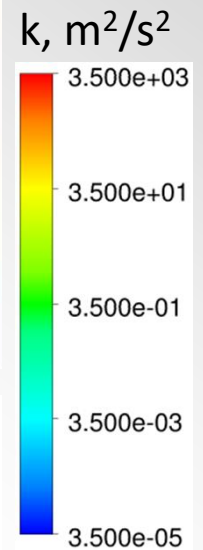
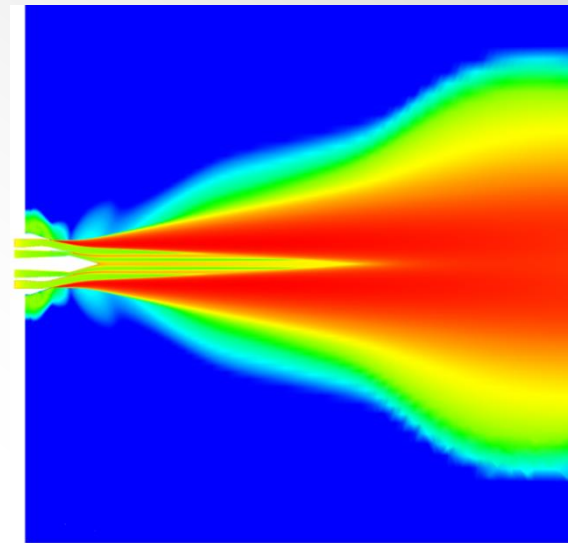
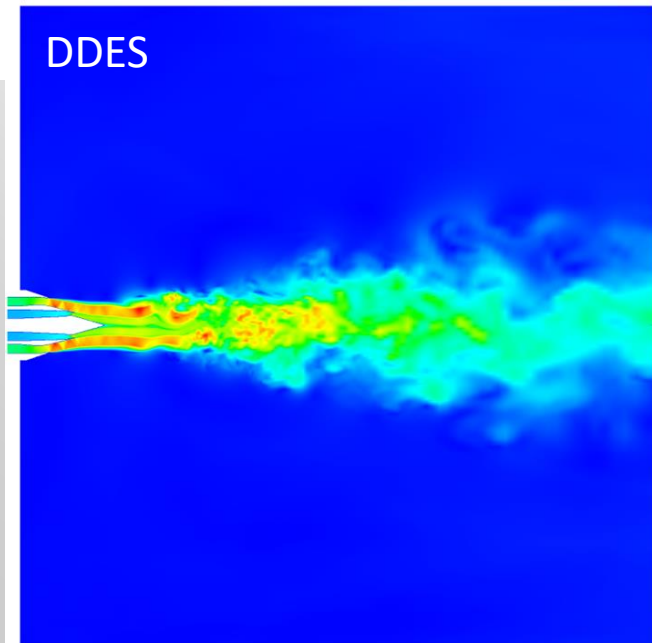
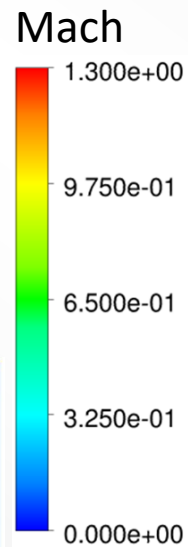
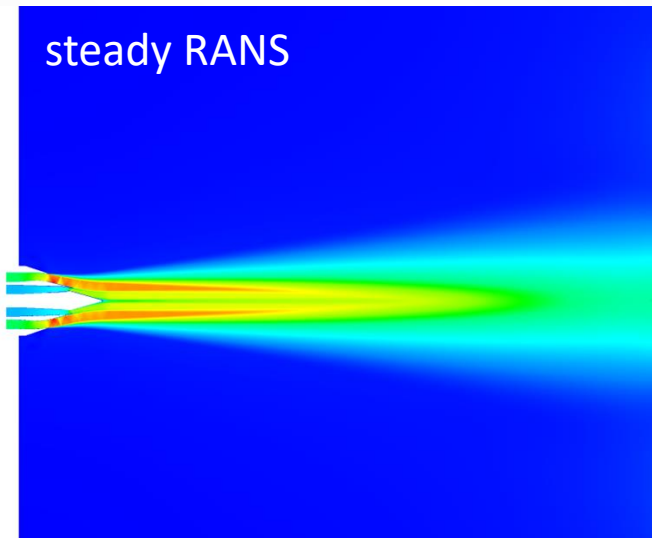
Meshes for finite volume computations



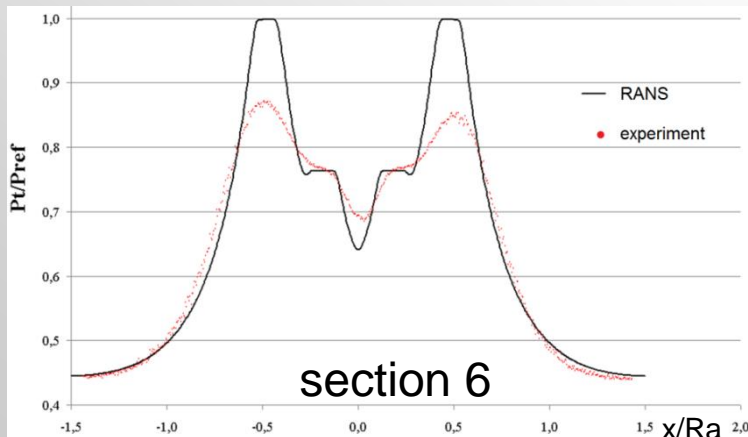
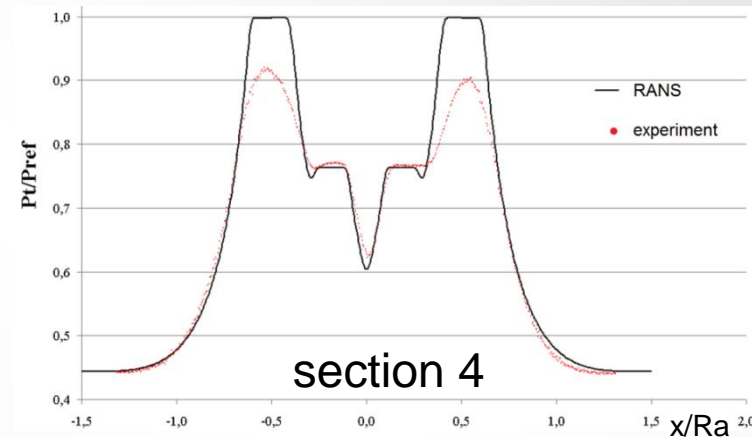
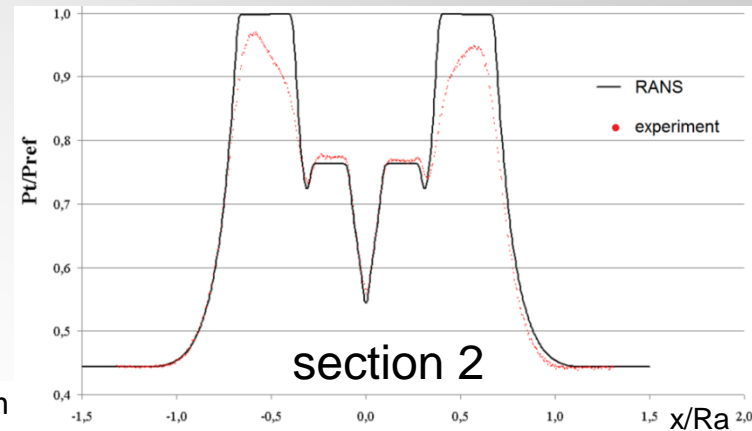
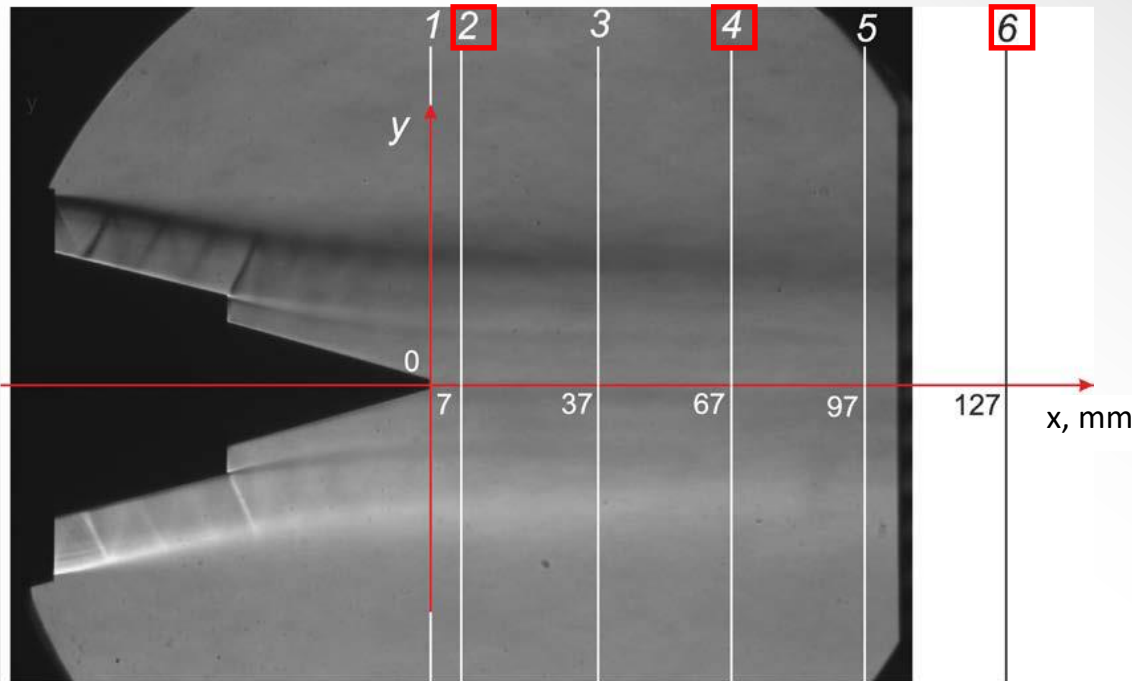
- Free outer boundaries with supersonic outlet
- Smooth adiabatic no-slip nozzle walls
- Inlet with uniform flow in the contours, $Tu=1\%$ (outer contour) and 10% (inner contour; decays quickly within the nozzle)

- 3D hexahedral structured mesh, $25 \cdot 10^6$ cells
- Initial steady RANS computation using SST turbulence model
- RANS flowfield used as initial conditions for unsteady DES-SST computation
- 140 characteristic times have been simulated

Flowfields obtained in computations

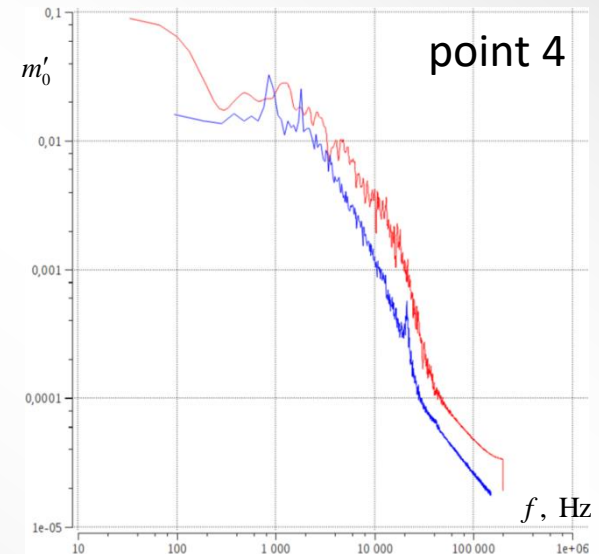
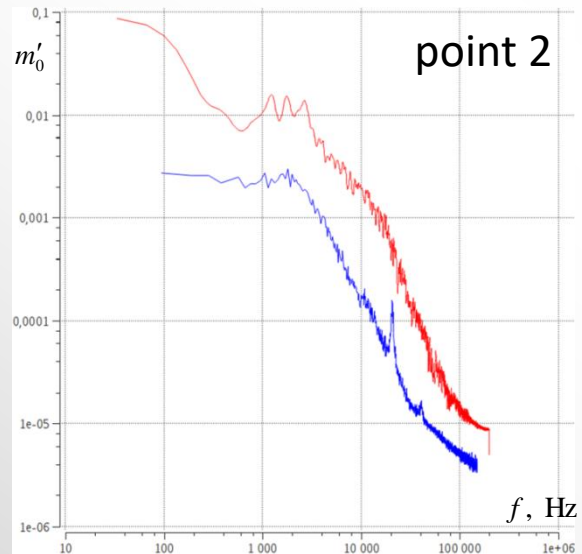
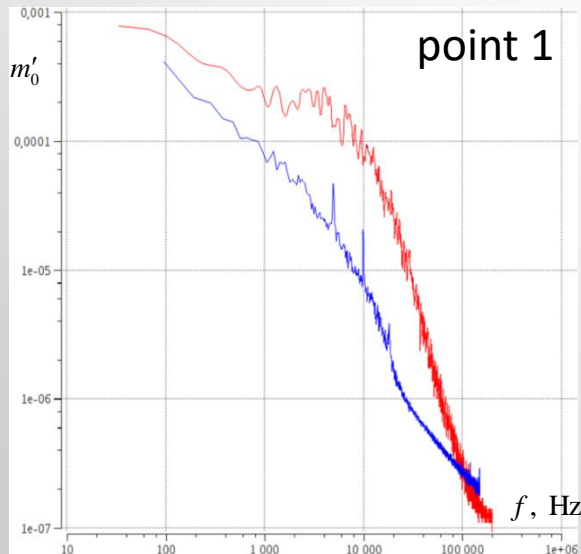
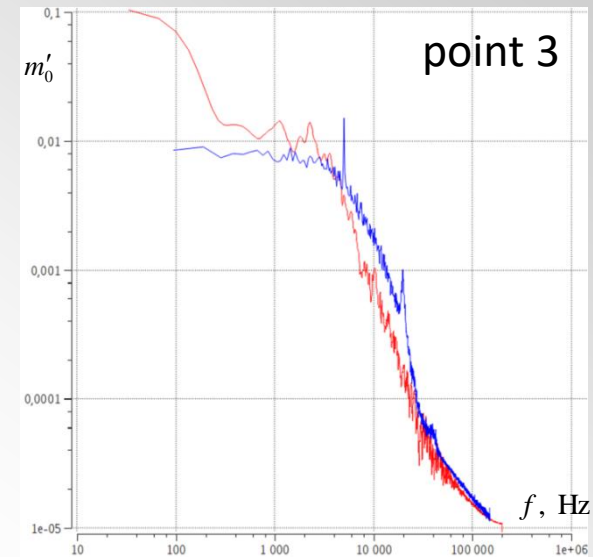
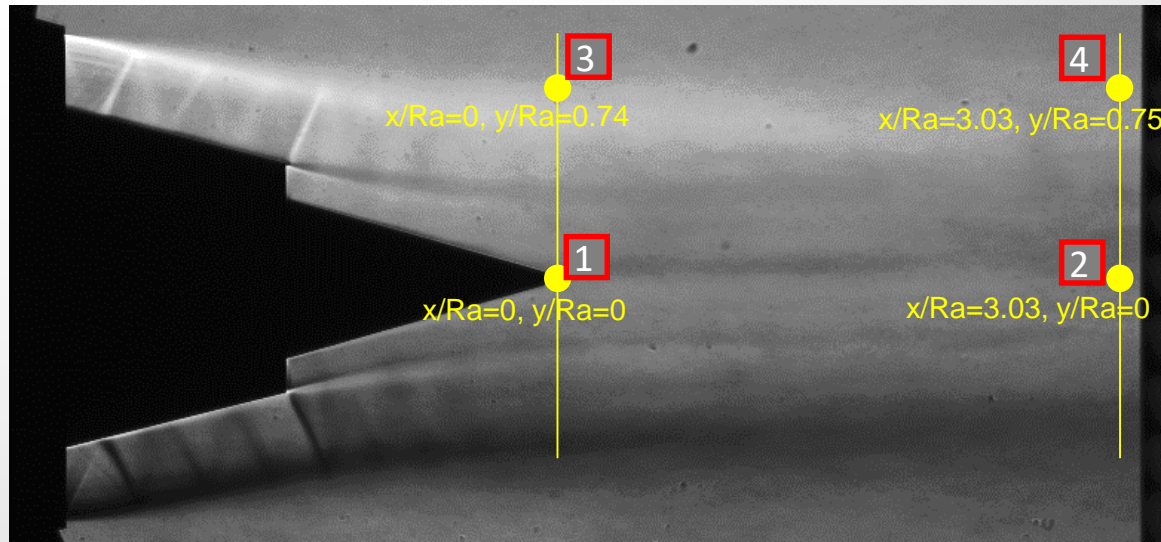


RANS computation results: pressure profiles



- boundary layers in outer contour are too thin in the computation
- wake diffusion behind the central body is underpredicted
- outer mixing layer growth rate is captured well

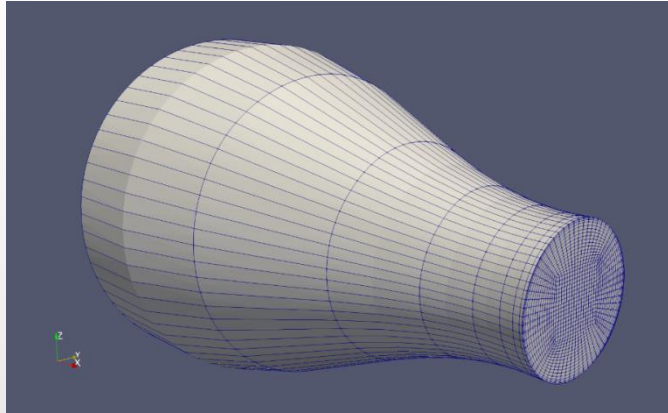
DES computation results: mass flow rate spectra



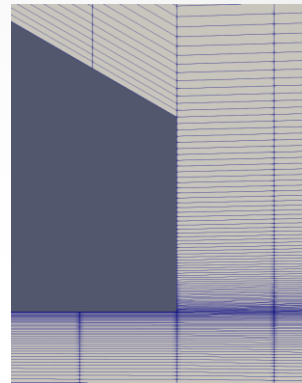
- spectra in shear layer are predicted better than along the centerline

Nozzle test case: mesh and first computations

Inner surface of nozzle (medium mesh)

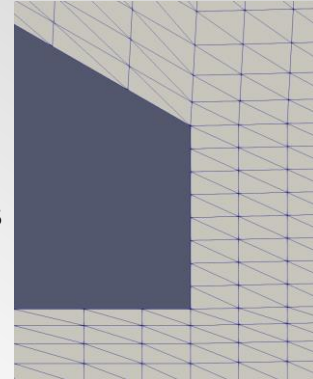


Nozzle tip



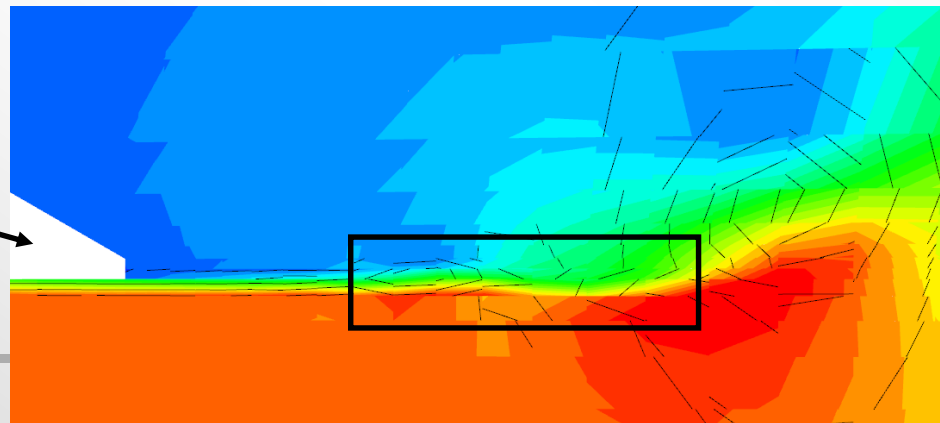
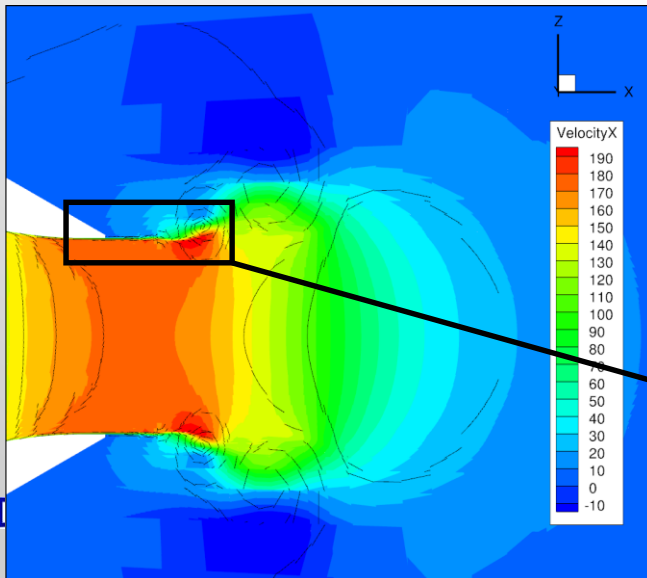
Fine mesh
from NASA
website

New fine
DDES
mesh for
wall
functions



Problems with Discontinuous Galerkin computations

- instability at the origin of the shear layer
- problem is independent of Mach number
- DG monotonization is now considered



Conclusions

- DG approach up to $K=5$ have implemented in TsAGI's CFD code successfully;
- To achieve enstrophy error lower than 20% in the Taylor–Green vortex problem, WENO class A scheme requires at least twice more time than high order DG. This difference becomes larger as the required accuracy grows;
- In the computations on a cluster of up to 4000 cores, the speed of the program is increased by more than 1.8 times with each doubling of core number. Use of a biggest number of cores requires a multilevel parallelization involving OpenMP;
- Second order FV RANS and DDES calculations for nozzle test case performed. For high-order DG calculations of nozzle limiting procedure is required;